

การชงกาแฟแบบปั๊มสุญญากาศ

โดย

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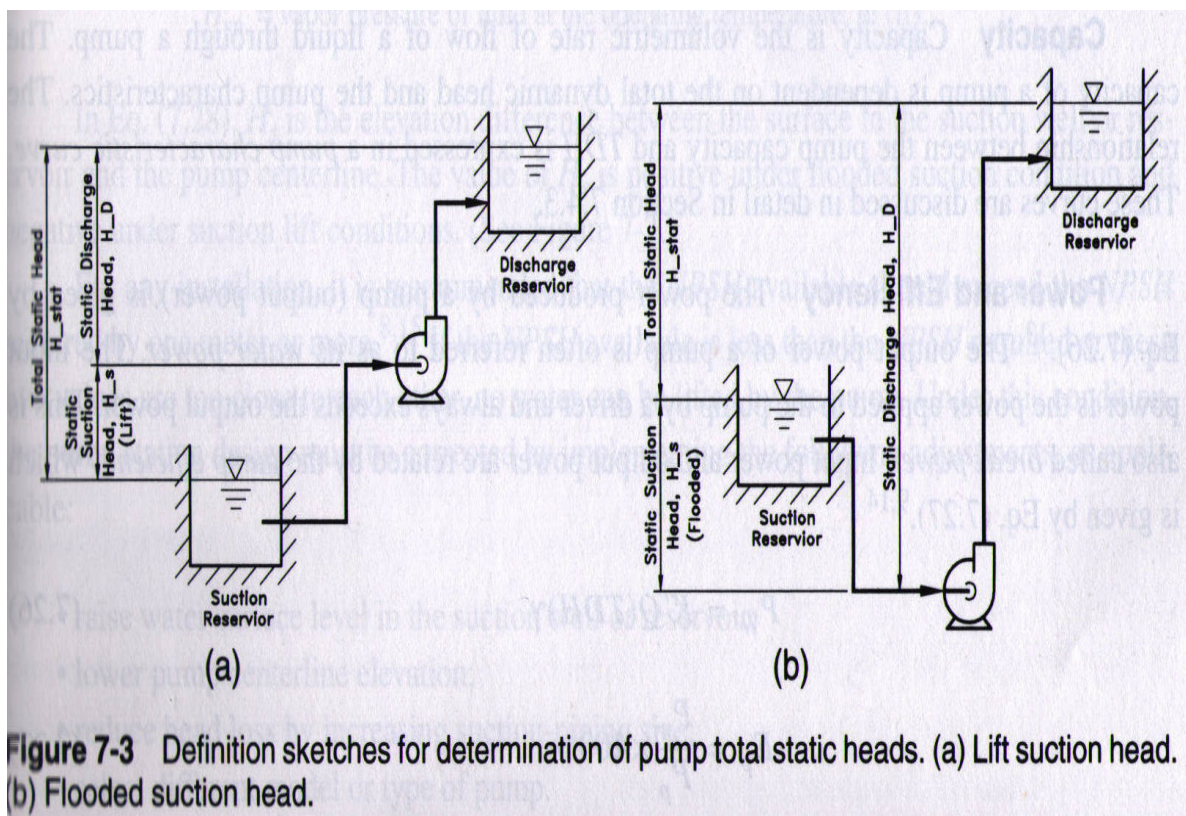
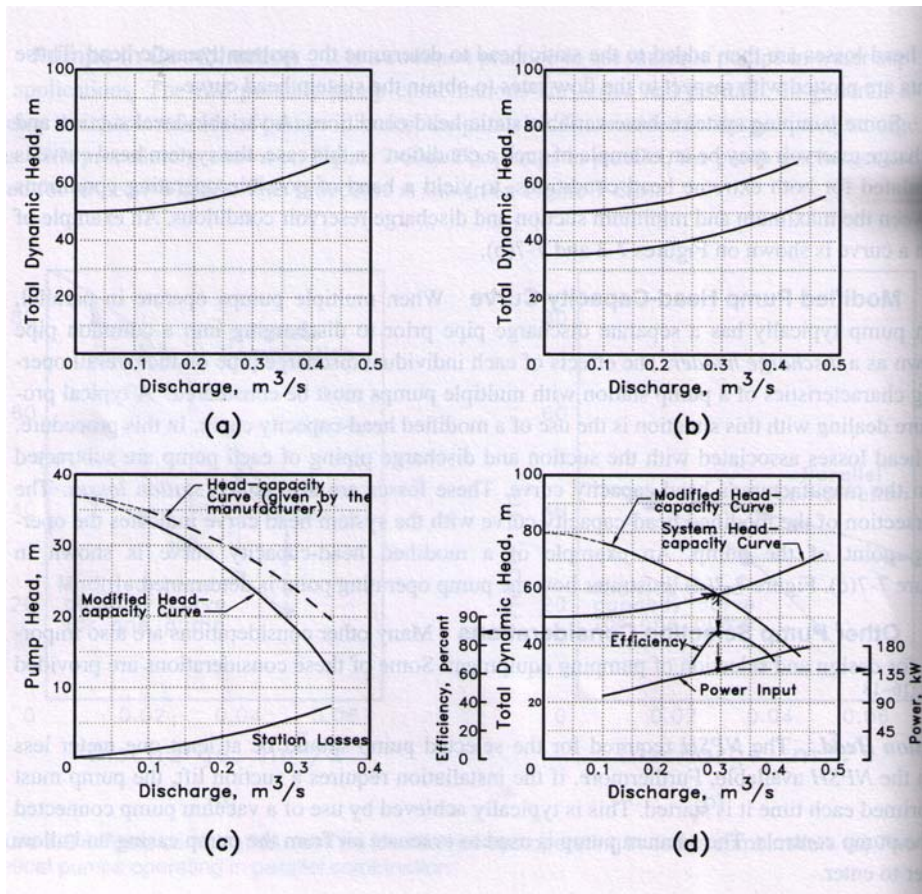


Figure 7-3 Definition sketches for determination of pump total static heads. (a) Lift suction head. (b) Flooded suction head.



Calculate Head Loss in Suction Pipe

1. Flow rate	=	15	m ³ /hr
2. Static Suction Lift	=	2	m. (pipe length)
3. Static Discharge Head	=	10	m. (pipe length)
4. Suction Pipe Diameter	=	2	in
	=	0.0508	m.
5. Discharge Pipe Diameter	=	2	in
	=	0.0508	m.

$$\text{Equation } Q = Av$$

$$A = \frac{\pi D^2}{4}$$

$$\therefore v = \frac{Q \times 4}{\pi D^2}$$

$$\therefore v = 7400.72 \text{ m/hr} = 2.055755 \text{ m/sec}$$

1. Calculate headloss in Suction Pipe

1.1 Minor losses due entrance

$$\text{Equation } h_m = K \frac{v^2}{2g}$$

where

h_m	=	headloss inlet pipe (m)
v	=	water velocity in suction pipe (m/s)
g	=	Acceleration due to gravity 9.81

Entrance	K
Pipe Project into tank	0.83
End of pipe flushed with tank	0.5
Slightly rounded	0.23
Bell mouthed	0.04
	Use
	0.5

$$\therefore h_m = 0.1077 \text{ m.}$$

1.2 Minor Head losses in Pressure Conduits

$$h = k \frac{v^2}{2g} \times \text{number}$$

Table 1 Minor Head losses in Pressure Conduits

Item	K factor	Number	Total Minor head loss
1. Gate Valve			
- Full open	0.19	1	0.040926
- One-fourth closed	1.15		0
- One - half closed	5.6		0
- Three - fourths closed	24		0
- Typical value	1		0
2. Butterfly Valve			
- Full open	0.3		0
- 20°	1.4		0
- Angle closed 40°	10		0
- 60°	94		0
- Typical value	1.2		0
3. Check valve			
- K = 1.5 - 2.5	1.5		0
4. Plug Valve	1		0
5. Elbow (45 -61 cm diameter)			
- 22.5° (K = 0.1 - 0.2)	0.1		0
- 45° (K = 0.2 - 0.3)	0.2		0
- 90° (K = 0.25 - 0.6)	0.25		0
6. Tee			
- Run to run (K = 0.25 - 0.6)	0.25		0
- Branch to run(K= 0.6 - 1.8)	0.6		0
- Run to branch(K=0.6 - 1.8)	0.6		0

Table 1 Minor Head losses in Pressure Conduits (continues)

Item	K factor	Number	Total Minor head loss
7. Reduer (with angle of divergence 10° - 20°) K = 0.15 - 0.2	0.15		0
8. Increaser (with angle of divergence 10° - 20°) K = 0.05 - 0.3	0.05		0
		Total	0.040926

1.3 Friction losses in the pipe are typically calculated from Dracy-Weisbach

$$\text{Equation } h_f = f \frac{L v^2}{D 2g}$$

where

- h_f = total friction head loss in suction or discharge pipes,(m)
- L = length of pipe,(m) (suction or discharge pipes)
- D = diameter of the pipe,(m)
- v = velocity in the pipe,(m/s)
- f = coefficient of friction (Dracy - Weisbach) page 636(integrated design of water treatment facilities)
- g = acceleration due to gravity, 9.81 m/s²

Table 2 Approximate Minor Head Losses in Fittings and Valves (page 98, water and wastewater technology)

Fitting and Valve	Equivalent Length (Diameters of pipe)	Loss Coefficient k
1. Tee (run)	20	0.6
2. Tee (branch)	60	1.8
3. 90° bend-		
Short radius	32	0.9
Medium radius	27	0.75
Long radius	20	0.6
4. 45° bend	15	0.42
5. Gate Valve (full open)	17	0.48
(open 1:4)	1000	
6. Swing check valve(open)	135	3.7
7. Butterfly valve (open)	40	1.2
8. Glove valve (open)	200	
9. Check valve(full open)	150	
10. Check valve with strainer	400	

Table 3 Approximate Minor Head Losses in Fittings and Valves in term Equivalent Pipe Length(L_e)

ออกแบบปั๊มสูบน้ำ(ผ่านสารกรอง2)head loss in suction pipe

Fitting and Valve	Equivalent Length (Diameters of pipe)	Pieces	Diameter suction pipe (m.)	Equivalent Pipe(L_e) (m.)
1. Tee (run)	20		0.0508	0
2. Tee (branch)	60		0.0508	0
3. 90° bend-				0
Short radius	32		0.0508	0
Medium radius	27		0.0508	0
Long radius	20		0.0508	0
4. 90° Standard	30	1	0.0508	1.524
4. 45° bend	15		0.0508	0
5. Gate Valve (full open)	17		0.0508	0
(open 1:4)	1000		0.0508	0
6. Swing check valve(open)	135		0.0508	0
7. Butterfly valve (open)	40	1	0.0508	2.032
8. Glove valve (open)	200		0.0508	0
9. Check valve(full open)	150		0.0508	0
10. Check valve with strainer	400		0.0508	0
Total (L_e)				3.556

$$\begin{aligned}
 \therefore \quad \text{Equivalent length in Suction pipe } (L_{\text{suc}}) &= \text{ suction pipe length} + L_e \\
 &= 2 + 3.556 \\
 L_{\text{suc}} &= 5.556
 \end{aligned}$$

Table 4 Relation between surface conditions and friction coefficient

ออกแบบปั๊มสูบน้ำ(ผ่านสารกรอง2) head loss in suction pipe

Condition	Friction Coefficient			Notes
	n (Manning's)	C (Hazen's)	f (Darcy's)	
Very smooth surface	0.01	140	0.0002	PVC Pipe clean cement lined pipe
Fair condition surface	0.013	120	0.0012	Unlined pipe
Rough surface	0.016	100	0.0025	Rusted pipe

Pipe Material : **PVC** \therefore f = **0.0002**

Theory

$$\text{Equation } h_f = f \frac{L v^2}{D 2g}$$

$$\therefore h_f = 0.004712 \text{ m.}$$

\therefore Total Head Loss in Suction Pipe : $h_m + \text{Minor head losses in Pressure Conduits} + h_f$

$$\text{Total Head Loss in Suction Pipe : } 0.153337$$

Calculate Head Loss in Discharge Pipe

1. Flow rate	=	15	m ³ /hr	=	0.004167	m ³ /s
2. Static Suction Lift	=	2	m.	(pipe length)		
3. Static Discharge Head	=	10	m.	(pipe length)		
4. Suction Pipe Diameter	=	2	in			
	=	0.0508	m.			
5. Discharge Pipe Diameter	=	2	in			
	=	0.0508	m.			

$$\text{Equation } Q = Av$$

$$A = \frac{\pi D^2}{4}$$

$$\therefore v = \frac{Q \times 4}{\pi D^2}$$

$$\therefore v = 7400.72 \text{ m/hr} = 2.055755 \text{ m/sec}$$

2. Calculate headloss in Discharge Pipe

2.1 Minor losses due exit

$$\text{Equation } h_m = K \frac{v^2}{2g}$$

Exit	K
conduit to still water	1

where

h_m = headloss inlet pipe (m)

v = water velocity in discharge pipe (m/s)

g = Acceleration due to gravity 9.81 m/s²

$$\therefore h_m = 0.215399 \text{ m.}$$

2.2 Minor Head losses in Pressure Conduits

$$h = k \frac{v^2}{2g} \times \text{number}$$

Table 5 Minor Head losses in Pressure Conduits

Item	K factor	Number	Total Minor head loss
1. Gate Valve			
- Full open	0.19	1	0.040926
- One-fourth closed	1.15		0
- One - half closed	5.6		0
- Three - fourths closed	24		0
- Typical value	1		0
2. Butterfly Valve			
- Full open	0.3		0
- 20°	1.4		0
- Angle closed 40°	10		0
- 60°	94		0
- Typical value	1.2		0
3. Check valve			
- K = 1.5 - 2.5	1.5		0
4. Plug Valve	1		0
5. Elbow (45 -61 cm diameter)			
- 22.5° (K = 0.1 - 0.2)	0.1		0
- 45° (K = 0.2 - 0.3)	0.2		0
- 90° (K = 0.25 - 0.6)	0.25		0
6. Tee			
- Run to run (K = 0.25 - 0.6)	0.25		0
- Branch to run(K= 0.6 - 1.8)	0.6		0
- Run to branch(K=0.6 - 1.8)	0.6		0

Table 5 Minor Head losses in Pressure Conduits (continues)

Item	K factor	Number	Total Minor head loss
7. Reducer (with angle of divergence 10° - 20°) K = 0.15 - 0.2	0.15		0
8. Increaser (with angle of divergence 10° - 20°) K = 0.05 - 0.3	0.05		0
		Total	0.040926

2.3 Friction losses in the pipe are typically calculated from Dracy-Weisbach

$$\text{Equation } h_f = f \frac{L v^2}{D 2g}$$

where

- h_f = total friction head loss in suction or discharge pipes,(m)
- L = length of pipe,(m) (suction or discharge pipes)
- D = diameter of the pipe,(m)
- v = velocity in the pipe,(m/s)
- f = coefficient of friction (Dracy - Weisbach) page 636(integrated design of water treatment facilities)
- g = acceleration due to gravity, 9.81 m/s²

Table 6 Approximate Minor Head Losses in Fittings and Valves (page 98, water and wastewater technology)

Fitting and Valve	Equivalent Length (Diameters of pipe)	Loss Coefficient k
1. Tee (run)	20	0.6
2. Tee (branch)	60	1.8
3. 90° bend-		
Short radius	32	0.9
Medium radius	27	0.75
Long radius	20	0.6
4. 45° bend	15	0.42
5. Gate Valve (full open)	17	0.48
(open 1:4)	1000	
6. Swing check valve(open)	135	3.7
7. Butterfly valve (open)	40	1.2
8. Glove valve (open)	200	
9. Check valve(full open)	150	
10. Check valve with strainer	400	

Table 7 Approximate Minor Head Losses in Fittings and Valves in term Equavalent Pipe Length(L_e)

ออกแบบปั๊มสูบน้ำ(ผ่านตารางรอง2)head loss in discharge pipe

Fitting and Valve	Equivalent Length (Diameters of pipe)	Pieces	Diameter suction pipe (m.)	Equivalent Pipe(L_e) (m.)
1. Tee (run)	20		0.0508	0
2. Tee (branch)	60		0.0508	0
3. 90° bend-				0
Short radius	32		0.0508	0
Medium radius	27		0.0508	0
Long radius	20		0.0508	0
4. 90° Standard	30	1	0.0508	1.524
4. 45° bend	15		0.0508	0
5. Gate Valve (full open)	17		0.0508	0
(open 1:4)	1000		0.0508	0
6. Swing check valve(open)	135		0.0508	0
7. Butterfly valve (open)	40	1	0.0508	2.032
8. Glove valve (open)	200		0.0508	0
9. Check valve(full open)	150		0.0508	0
10. Check valve with strainer	400		0.0508	0
Total (L_e)				3.556

$$\begin{aligned}
 \therefore \quad \text{Equivalent length in Suction pipe } (L_{\text{suc}}) &= \text{ suction pipe length} + L_e \\
 &= 2 + 3.556 \\
 L_{\text{suc}} &= 5.556
 \end{aligned}$$

Table 8 Relation between surface conditions and friction coefficient

ออกแบบปั๊มสูบน้ำ(ผ่านตารางรอง2) head loss in discharge pipe

Condition	Friction Coefficient			
	n (Manning's)	C (Hazen's)	f (Darcy's)	Notes
Very smooth surface	0.01	140	0.0002	PVC Pipe clean cement lined pipe
Fair condition surface	0.013	120	0.0012	Unlined pipe
Rough surface	0.016	100	0.0025	Rusted pipe

Pipe Material : **PVC** ∴ f = **0.0002**

Theory

$$\text{Equation } h_f = f \frac{L v^2}{D 2g}$$

∴ $h_f = 0.004712 \text{ m.}$

∴ Total Head Loss in Discharge Pipe = $h_m + \text{Minor head losses in Pressure Conduits} + h_f$

Total Head Loss in Discharge Pipe = 0.2610366

Calculate Head Loss in Influent Pipe

1. Flow rate	=	15	m ³ /hr	=	0.004167	m ³ /s
2. Static Suction Lift	=	2	m.	(pipe length)		
3. Static Discharge Head	=	10	m.	(pipe length)		
4. Suction Pipe Diameter	=	2	in			
	=	0.0508	m.			
5. Discharge Pipe Diameter	=	2	in			
	=	0.0508	m.			

$$\text{Equation } Q = Av$$

$$A = \frac{\pi D^2}{4}$$

$$\therefore v = \frac{Q \times 4}{\pi D^2}$$

$$\therefore v = 7400.72 \text{ m/hr} = 2.055755 \text{ m/sec}$$

3. Calculate headloss in Inlet Pipe

3.1 Minor losses due entrance

$$\text{Equation } h_m = K \frac{v^2}{2g}$$

Entrance	K
conduit to still water	0.5

where

h_m = headloss inlet pipe (m)

v = water velocity in entrance pipe (m/s)

g = Acceleration due to gravity 9.81 m/s²

$$\therefore h_m = 0.1077 \text{ m.}$$

3.2 Minor Head losses in Pressure Conduits

$$h = k \frac{v^2}{2g} \times \text{number}$$

Table 9 Minor Head losses in Pressure Conduits

Item	K factor	Number	Total Minor head loss
1. Gate Valve			
- Full open	0.19	1	0.040926
- One-fourth closed	1.15		0
- One - half closed	5.6		0
- Three - fourths closed	24		0
- Typical value	1		0
2. Butterfly Valve			
- Full open	0.3		0
- 20°	1.4		0
- Angle closed 40°	10		0
- 60°	94		0
- Typical value	1.2		0
3. Check valve			
- K = 1.5 - 2.5	1.5		0
4. Plug Valve	1		0
5. Elbow (45 -61 cm diameter)			
- 22.5° (K = 0.1 - 0.2)	0.1		0
- 45° (K = 0.2 - 0.3)	0.2		0
- 90° (K = 0.25 - 0.6)	0.25		0
6. Tee			
- Run to run (K = 0.25 - 0.6)	0.25		0
- Branch to run(K= 0.6 - 1.8)	0.6		0
- Run to branch(K=0.6 - 1.8)	0.6		0

Table 9 Minor Head losses in Pressure Conduits (continues)

Item	K factor	Number	Total Minor head loss
7. Reducer (with angle of divergence 10° - 20°) K = 0.15 - 0.2	0.15		0
8. Increaser (with angle of divergence 10° - 20°) K = 0.05 - 0.3	0.05		0
		Total	0.040926

3.3 Friction losses in the pipe are typically calculated from Dracy-Weisbach

$$\text{Equation } h_f = f \frac{L v^2}{D 2g}$$

where

- h_f = total friction head loss in suction or discharge pipes,(m)
- L = length of pipe,(m) (suction or discharge pipes)
- D = diameter of the pipe,(m)
- v = velocity in the pipe,(m/s)
- f = coefficient of friction (Dracy - Weisbach) page 636(integrated design of water treatment facilities)
- g = acceleration due to gravity, 9.81 m/s²

Table 10 Approximate Minor Head Losses in Fittings and Valves (page 98, water and wastewater technology)

Fitting and Valve	Equivalent Length (Diameters of pipe)	Loss Coefficient k
1. Tee (run)	20	0.6
2. Tee (branch)	60	1.8
3. 90° bend-		
Short radius	32	0.9
Medium radius	27	0.75
Long radius	20	0.6
4. 45° bend	15	0.42
5. Gate Valve (full open)	17	0.48
(open 1:4)	1000	
6. Swing check valve(open)	135	3.7
7. Butterfly valve (open)	40	1.2
8. Glove valve (open)	200	
9. Check valve(full open)	150	
10. Check valve with strainer	400	

Table 11 Approximate Minor Head Losses in Fittings and Valves in term Equavalent Pipe Length(L_e)

ออกแบบปั๊มสูบน้ำ(ผ่านตารางของ2)head loss Influent pipe

Fitting and Valve	Equivalent Length (Diameters of pipe)	Pieces	Diameter suction pipe (m.)	Equivalent Pipe(L_e) (m.)
1. Tee (run)	20		0.0508	0
2. Tee (branch)	60		0.0508	0
3. 90° bend-				0
Short radius	32		0.0508	0
Medium radius	27		0.0508	0
Long radius	20		0.0508	0
4. 90° Standard	30	1	0.0508	1.524
4. 45° bend	15		0.0508	0
5. Gate Valve (full open)	17		0.0508	0
(open 1:4)	1000		0.0508	0
6. Swing check valve(open)	135		0.0508	0
7. Butterfly valve (open)	40	1	0.0508	2.032
8. Glove valve (open)	200		0.0508	0
9. Check valve(full open)	150		0.0508	0
10. Check valve with strainer	400		0.0508	0
Total (L_e)				3.556

$$\begin{aligned}
 \therefore \quad \text{Equivalent length in Suction pipe } (L_{\text{suc}}) &= \text{ suction pipe length} + L_e \\
 &= 2 + 3.556 \\
 L_{\text{suc}} &= 5.556
 \end{aligned}$$

Table 12 Relation between surface conditions and friction coefficient

ออกแบบปั๊มสูบน้ำ(ผ่านตารางรอง2) head loss Influent pipe

Condition	Friction Coefficient			
	n (Manning's)	C (Hazen's)	f (Darcy's)	Notes
Very smooth surface	0.01	140	0.0002	PVC Pipe clean cement lined pipe
Fair condition surface	0.013	120	0.0012	Unlined pipe
Rough surface	0.016	100	0.0025	Rusted pipe

Pipe Material : **PVC** ∴ f = **0.0002**

Theory

$$\text{Equation } h_f = f \frac{L v^2}{D 2g}$$

∴ $h_f = 0.004712 \text{ m.}$

∴ Total Head Loss in Influent Pipe = $h_m + \text{Minor head losses in Pressure Conduits} + h_f$

Total Head Loss in Influent Pipe = 0.153337

Calculate Head Loss Through Media

4 Choose Single filter media and Backwash by pump

Flow Rate = 15 m³/hr

Number of Filtration Tank = 3 Tank

Flow Rate per Tank = 5 m³/hr

∴ Flow Rate Per Tank for Design = 8 m³/hr

Calculate Entrance Pipe Size

Give Water Velocity = 0.60 m/s

From $d = \sqrt{\frac{4Q}{\pi v}}$

∴ $d = 0.067$ m.

or = 100 mm.

Calculate dimensional requirements (circular basin)

Design Criteria Filtration rate = 5-10 m³/hr/m²

Chosed = 5 m³/hr/m²

∴ Surface Area for each Filtration Tank = 1.5 m²

Diameter Basin = 1.382 m

Acture Diameter = 1.4 m

Choose Backwash Rate = 0.8 m/min

Backwash Times = 10 min

Backwash/Day = 1 Time

Water Loss = 11 m³/day

Flow Rate = 8.0 m³/hr

Acture Filtration Rate = 5.3 m³/hr/m²

Acture Filtration Rate if Backwash 1 Tank = 8.0 m³/hr/m²

or Flow Rate per Tank if Backwash 1 Tank = 12 m³/hr

Chosed Media

Sand Media

- Effective size 0.45 - 0.65 mm. average 0.5 mm.
- Uniformity coefficient 1.40 - 1.70
- Depth 0.65 m

Anthracite Media

- Effective size 0.7 - 2 mm. average 1.35 mm.
- Uniformity coefficient 1.3 - 1.8
- Depth 0.3 - 0.6 m.

$$\text{Equation } N_R = \frac{D_p \rho_L v}{\mu}$$

Where N_R = Renolds number

D_p = Media grain diameter,(m)

ρ_L = Density of water, (kg/m³) 25° C = 997.1 kg/m³

v = filtration velocity,(m/s)

μ = absolute viscosity, N-s/m²(kg.m/s²)

25° C = 0.0009 N-s/m²

$$\text{Equation } f = 150 \frac{(1-e)}{N_R} + 1.75$$

Where f = friction factor

e = porosity ratio (usually 0.4 - 0.5)

N_R = Renolds number

$$\text{Equation } \text{Carmen - Kozeny } h_L = \frac{f (1-e) L v^2}{\phi e^3 d g}$$

Where h_L = head loss, (m)

f = friction factor

e = porosity ratio (usually 0.4 - 0.5)

L = media depth,(m)

d = media grain diameter,(m)

v = filtration velocity,(m/s)

g = acceleration due to gravity 9.81 m/s^2

ϕ = particle shape factor (usually 0.85 to 1) 1

4.1 Calculations of Head Loss through Clean Filter Media

Layer	Size,(mm)	Depth,(m)	Porosity (e)	Reynolds Number N_R	Friction Coefficient f	Head Loss h_L (m)
Anthracite Layer	1.0	0.5	0.48	1.64361	49.2065	0.02566665
Sand Layer	0.5	0.25	0.4	0.8218	111.265	0.115716941
Total Depth of the media layer		0.75	Total head loss through clean filter media			0.141383591

4.2 Calculations of Head Loss through the Gravel Support

Layer	Size,(mm)	Depth,(m)	Porosity (e)	Reynolds Number N_R	Friction Coefficient f	Head Loss h_L (m)
Top Layer	1.0	0.5	0.45	1.64361	51.9444	0.03478009
Second Layer	1.0	0.5	0.45	1.64361	51.9444	0.03478009
Third Layer	0.5	0.25	0.40	0.8218	111.265	0.11571694
Bottom Layer	0.5	0.25	0.40	0.8218	111.265	0.11571694
Total Depth of the gravel layer		0.75	Total head loss through the gravel layer			0.150497036

4.3 Calculations of Head Loss through the underdrain system

$$\text{Equation } H_L = k_1 v^2$$

where

H_L = head loss through the underdrain, m

k_1 = head loss constant that varies with the type of underdrain system
(generally is given by the manufacture)

(for the tile underdrain selected $k_1 = 0.0005$)

v = filtration velocity, m/hr

$$\therefore H_L = 0.0141 \text{ m.}$$

Total Head loss across the filter Media at Q_{design}

= Head Loss Through Clean Filter + Head Loss Through the Gravel Support + Head Loss the Underdrain

= 0.291880627 m.

Calculate Head Loss in Effluent Pipe

1. Flow rate	=	15	m ³ /hr	=	0.004167	m ³ /s
2. Static Suction Lift	=	2	m.	(pipe length)		
3. Static Discharge Head	=	10	m.	(pipe length)		
4. Suction Pipe Diameter	=	2	in			
	=	0.0508	m.			
5. Discharge Pipe Diameter	=	2	in			
	=	0.0508	m.			

$$\text{Equation } Q = Av$$

$$A = \frac{\pi D^2}{4}$$

$$\therefore v = \frac{Q \times 4}{\pi D^2}$$

$$\therefore v = 7400.72 \text{ m/hr} = 2.055755 \text{ m/sec}$$

3. Calculate headloss in Inlet Pipe

3.1 Minor losses due entrance

$$\text{Equation } h_m = K \frac{v^2}{2g}$$

Entrance	K
conduit to still water	0.5

where

h_m = headloss inlet pipe (m)

v = water velocity in entrance pipe (m/s)

g = Acceleration due to gravity 9.81 m/s²

$$\therefore h_m = 0.1077 \text{ m.}$$

3.2 Minor Head losses in Pressure Conduits

$$h = k \frac{v^2}{2g} \times \text{number}$$

Table 9 Minor Head losses in Pressure Conduits

Item	K factor	Number	Total Minor head loss
1. Gate Valve			
- Full open	0.19	1	0.040926
- One-fourth closed	1.15		0
- One - half closed	5.6		0
- Three - fourths closed	24		0
- Typical value	1		0
2. Butterfly Valve			
- Full open	0.3		0
- 20°	1.4		0
- Angle closed 40°	10		0
- 60°	94		0
- Typical value	1.2		0
3. Check valve			
- K = 1.5 - 2.5	1.5		0
4. Plug Valve	1		0
5. Elbow (45 -61 cm diameter)			
- 22.5° (K = 0.1 - 0.2)	0.1		0
- 45° (K = 0.2 - 0.3)	0.2		0
- 90° (K = 0.25 - 0.6)	0.25		0
6. Tee			
- Run to run (K = 0.25 - 0.6)	0.25		0
- Branch to run(K= 0.6 - 1.8)	0.6		0
- Run to branch(K=0.6 - 1.8)	0.6		0

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Table 9 Minor Head losses in Pressure Conduits (continues)

Item	K factor	Number	Total Minor head loss
7. Reduer (with angle of divergence 10° - 20°) K = 0.15 - 0.2	0.15		0
8. Increaser (with angle of divergence 10° - 20°) K = 0.05 - 0.3	0.05		0
		Total	0.040926

3.3 Friction losses in the pipe are typically calculated from Dracy-Weisbach

$$\text{Equation } h_f = f \frac{L v^2}{D 2g}$$

where

- h_f = total friction head loss in suction or discharge pipes,(m)
 L = length of pipe,(m) (suction or discharge pipes)
 D = diameter of the pipe,(m)
 v = velocity in the pipe,(m/s)
 f = coefficient of friction (Dracy - Weisbach) page 636(integrated design of water treatment facilities)
 g = acceleration due to gravity, 9.81 m/s²

Table 10 Approximate Minor Head Losses in Fittings and Valves (page 98, water and wastewater technology)

Fitting and Valve	Equivalent Length (Diameters of pipe)	Loss Coefficient k
1. Tee (run)	20	0.6
2. Tee (branch)	60	1.8
3. 90° bend-		
Short radius	32	0.9
Medium radius	27	0.75
Long radius	20	0.6
4. 45° bend	15	0.42
5. Gate Valve (full open)	17	0.48
(open 1:4)	1000	
6. Swing check valve(open)	135	3.7
7. Butterfly valve (open)	40	1.2
8. Glove valve (open)	200	
9. Check valve(full open)	150	
10. Check valve with strainer	400	

Table 11 Approximate Minor Head Losses in Fittings and Valves in term Equivalent Pipe Length(L_e)

Fitting and Valve	Equivalent Length (Diameters of pipe)	Pieces	Diameter suction pipe (m.)	Equivalent Pipe(L_e) (m.)
1. Tee (run)	20		0.0508	0
2. Tee (branch)	60		0.0508	0
3.90° bend-				0
Short radius	32		0.0508	0
Medium radius	27		0.0508	0
Long radius	20		0.0508	0
4. 90° Standard	30	1	0.0508	1.524
4. 45° bend	15		0.0508	0
5. Gate Valve (full open)	17		0.0508	0
(open 1:4)	1000		0.0508	0
6. Swing check valve(open)	135		0.0508	0
7. Butterfly valve (open)	40	1	0.0508	2.032
8. Glove valve (open)	200		0.0508	0
9. Check valve(full open)	150		0.0508	0
10. Check valve with strainer	400		0.0508	0
Total (L_e)				3.556

$$\begin{aligned}
 \therefore \quad \text{Equivalent length in Suction pipe } (L_{\text{suc}}) &= \text{ suction pipe length} + L_e \\
 &= 2 + 3.556 \\
 L_{\text{suc}} &= 5.556
 \end{aligned}$$

Table 12 Relation between surface conditions and friction coefficient

ออกแบบปั๊มสูบน้ำ(ผ่านสารกรอง2)head loss Effluent pipe

Condition	Friction Coefficient			Notes
	n (Manning's)	C (Hazen's)	f (Darcy's)	
Very smooth surface	0.01	140	0.0002	PVC Pipe clean cement lined pipe
Fair condition surface	0.013	120	0.0012	Unlined pipe
Rough surface	0.016	100	0.0025	Rusted pipe

Pipe Material : **PVC** \therefore f = **0.0002**

Theory

$$\text{Equation } h_f = f \frac{L v^2}{D 2g}$$

$$\therefore h_f = 0.004712 \text{ m.}$$

$$\therefore \text{Total Head Loss in Effluent Pipe} = h_m + \text{Minor head losses in Pressure Conduits} + h_f$$

$$\text{Total Head Loss in Effluent Pipe} = 0.153337$$

3. Total Dynamic Head(TDH)

1. Head Loss in Suction Pipe = 0.153337
 2. Head Loss in Discharge Pipe = 0.261037
 3. Head Loss in Influent Pipe = 0.153337
 4. Head Loss Through Media = 0.291881
 5. Head Loss in Effluent Pipe = 0.153337
- ∴ Head Loss (pipe + media) = 1 + 2 + 3 + 4 + 5
1.012928
6. Total Static Head = 12

$$\text{Equation } \text{Total Dynamic Head} = \text{Total Static Head} + \text{Headloss (pipe + media)}$$

$$\therefore \text{Total Dynamic Head} = 13.01293 \text{ m.}$$

4. Power Requirement (Theory) or power output of the pump (water power)

$$\text{Equation } P_w = Q(m^3 / s) \times TDH (m) \times \gamma (KN / m^3) \text{ KW}$$

$$\text{Equation } P_w = Q(m^3 / s) \times TDH (m) \times 9.81 (KN / m^3)$$

$$\text{Equation } P_w = \frac{Q(ft^3 / s) \times TDH(ft) \times 62.4 lb / ft^3}{550} \text{ HP}$$

$$P_w = 0.531903 \text{ KN.m/s} \quad \text{Watt} = \text{N.m/s} = \text{kg.m/s}^2 \cdot \text{m/s}$$

$$\therefore P_w = 0.531903 \text{ KW}$$

5. Pump Power Requirement

$$\text{Equation } P_p = \frac{P_w}{\eta}$$

where

$$P_p = \text{Pump Power Requirement, (KW)}$$

Efficiency of pump and motor (50% - 60%) Choose $\eta = 60\%$
(Usually 70 - 90%)

$$\therefore P_p = 0.886506 \text{ KW}$$

$$\text{From } 0.7457 \text{ KW} = 1 \text{ HP}$$

$$\therefore 0.886506 \text{ KW} = 1.188824 \text{ HP}$$

Table 7-8 Typical Electrical Motor Efficiencies

Motor Power Rating, kW	Typical Efficiency, percent
1-5	70-80
5-7.5	80-85
7.5-20	85-88
20 and above	88-92

Source: Reference 26.

$$\text{Equation : } \rho = \frac{\gamma(N/m^3)}{g(m/s^2)}$$

$$\text{Equation : } F = m(kg) \times a(m/s^2)$$

$$\therefore F = ma \quad (kg \cdot m / s^2 = N)$$

$$\therefore \rho = \frac{\gamma(kg \cdot m / s^2) / m^3}{g(m/s^2)} \quad (kg / m^3)$$

Equation :

$$\therefore \gamma = \rho g \quad (N / m^3)$$

where

$$\rho_{H_2O} = 1000 \text{ kg/m}^3$$

$$g = 9.81 \text{ m/s}^2$$

$$\therefore \gamma = 1000(kg/m^3) \times 9.81(m/s^2)$$

$$\therefore \gamma = 9.81 \text{ K}(kg \cdot m / s^2) / m^3$$

$$\therefore \gamma = 9.81 \text{ KN/m}^3$$